

Book Chapter

A Comparative Energy and Economic Analysis between a Low Enthalpy Geothermal Design and Gas, Diesel and Biomass Technologies for a HVAC System Installed in an Office Building

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Abstract

This paper presents an analysis of economic and energy between a ground-coupled heat pump system and other available technologies, such as natural gas, biomass, and diesel, providing heating, ventilation, and air conditioning to an office building. All the proposed systems are capable of reaching temperatures of 22 °C/25 °C in heating and cooling modes. EnergyPlus software was used to develop a simulation model and carry out the validation process. The first objective of the paper is the validation of the numerical model developed in EnergyPlus with the experimental results collected from the monitored building to evaluate the system in other operating conditions and to compare

it with other available technologies. The second aim of the study is the assessment of the position of the low enthalpy geothermal system proposed versus the rest of the systems, from energy, economic, and environmental aspects. In addition, the annual heating and cooling seasonal energy efficiency ratio (COP_{sys}) of the ground-coupled heat pump (GCHP) shown is higher than the others. The economic results determine a period between 6 and 9 years for the proposed GCHP system to have lower economic cost than the rest of the systems. The results obtained determine that the GCHP proposed system can satisfy the thermal demand in heating and cooling conditions, with optimal environmental values and economic viability.

Keywords

Low Enthalpy Geothermal System; Renewable Energy; Simulation Models; Energy Analysis; Economic Feasibility

Nomenclature

BHE-Borehole Heat Exchanger; CO_2 -Fossil Carbon Dioxide Emissions (KG); COP_{SYS} -Seasonal Energy Efficiency Ratio; C_{ps} -Factor for Electricity (kg CO_2 /kWh); CTF_s -Conduction Transfer Method ; DHW-Domestic Hot Water; GCHP-Ground-Coupled Heat Pump; GHE-Ground Heat Exchanger; GLHEPRO-Ground Heat Exchanger Software; HVAC-Heating, Ventilating, and Air-Conditioning; MV-Mechanical Ventilation System; P_r -Dew Point Temperature ($^{\circ}C$); RF-Radiant Floor; R^2 -Coeff of Multiple Determinations; t_{ext} -Outdoor Air Temperature ($^{\circ}C$); t_{hw} -Hot-Water Temperature ($^{\circ}C$); t_{int} -Indoor Air Temperature ($^{\circ}C$); TRNSYS-Transient Systems Simulation; t_{sr} -Surface Average Temperature ($^{\circ}C$)

Introduction

Geothermal energy is recognized as a source of renewable energy that is environmentally friendly and technically feasible. For this reason, geothermal energy technologies can benefit from any climate mitigation policies. Directive 2009/28/EC [1] on the

promotion of the use of energy from renewable sources has been the most significant piece of EU legislation for geothermal energy. The main objectives are focused in the reduction of at least 20% in greenhouse gas (GHG) emissions compared to 1990 levels, with 20% of the final energy consumption to come from renewable sources and the improvement of energy efficiency by 20% compared to 2007 projections. These goals are the main drivers for the growth of geothermal technologies.

For this reason, development of low energy consumption in HVAC systems is important. Taking the year 2010 in Spain as reference, the main distribution of energy consumption in buildings is concentrated in HVAC, domestic hot water (DHW), equipment, and lighting. The HVAC and DHW necessities cover 62% of the total energy consumption in buildings [2]. The experimental installation built is composed of a ground-coupled heat pump (GCHP) and radiant floor (RF) system supported by a mechanical ventilation system (MV) [3,4]. The energy is extracted from a ground heat exchanger (GHE) drilled to a depth of 100 m [5,6,7].

There are many studies that analyze the behavior of each of the elements that make up the geothermal system, focusing on the operation of the GHE from theoretical and simulation models [8,9] or in an experimental installation monitoring different variables [10-13]. Related with control systems of the installation [14], demonstrate that the use of automatic systems to control the circulating pump speed versus classical adjustment case increased the coefficient of performance of the seasonal energy efficiency ratio (COP_{sys}) (COP_{sys} 7–8% higher and 7.5–8% lower CO_2 emission level).

Regarding the use of simulation tools [15], analyzed the results obtained and compared this with the experimental data, based on the development of two numerical simulation models using transient systems simulation (TRNSYS). The main parameters analyzed were the useful thermal energy and COP_{sys} . The results present a small difference in the COP_{sys} (4.5%) between radiator system and radiant floor. Using the same operating conditions, the radiator system has 10% higher energy consumption and CO_2

emissions than the radiant floor system. Using ground heat exchanger software (GLHEPRO) and TRANSYS software [16], analyzed the performance of an office building with heating and cooling necessities. The heat pump model development presented a deviation of 2% with respect to experimental results. The energy consumption of the heat pump unit was well predicted in the TRNSYS simulation model.

In relation to economic analysis [17], performed an economic analysis of a residential ground heat system in sedimentary rock formation, demonstrating the feasibility of the system from performance analysis and giving solutions for improvement with regard to a reduction in utility bills. The analysis done between GCHP and natural gas concludes that the data also indicate a marginal reduction in utility bill, and they center the improvements in operations strategies and optimal design, two aspects that are considered in this paper from the beginning.

The present study is characterized by analyzing a building located in Madrid, Spain [18], and consists of a GCHP, RF, and mechanical ventilation system to ensure the indoor air quality, not only focusing on energy performance, but also introducing the cost function and comparing the energy and economic results with other available technologies. The results obtained are used to analyze the possible investment for this type of installation from energy, economic, and environmental aspects, three principal factors to consider in any project. This new methodology analyzes the building globally. In addition, the difference to other studies is based on the action of four key points at the same time: decreasing the building energy demand, improving the energy efficiency of the building in the experimental design, developing a model simulation process to ensure the correct functioning of the installation, and validating the model designed in order to have a tool to simulate any building under different conditions. Using this model, a simulation process was done for the available technologies under the same building, obtaining energy results used to compare and analyze the viability of the GCHP system proposed. A new factor in this study is the comparative economic analysis between the technologies' comments, obtaining economic results

which determine the feasibility of the proposed design in a global way.

Experimental Building Description

Building Description

The office is located in Madrid, Spain, latitude $+40.7^\circ$ and longitude -3.99° and altitude of 1075 meters above sea level in a continental temperature climate (Figure 1a). The building has 189 m^2 floor area and includes a reception, office room, meeting room, archive room, rest room, bathroom, and experimental showroom (Figure 1b). The building is used as an office and is occupied by three people. The office building is located in Madrid, Spain, in a continental temperature climate (Figure 1). The indoor temperature design is $22/25^\circ \text{C}$ for heating and cooling modes, respectively. The building is equipped with a vertical U-tube GHE, GCHP, RF, and MV systems.

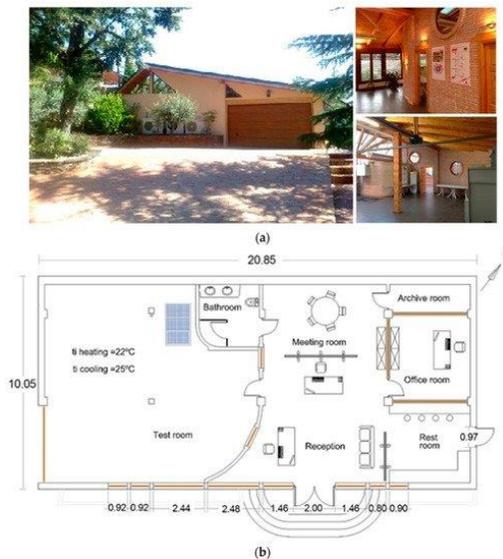


Figure 1: (a) Experimental office building; (b) Office floor plan.

As shown in Figure 2, the main parts of the system are BHE, heat pump unit, circulating water pumps, radiant floor, data acquisition instruments, and auxiliary parts.

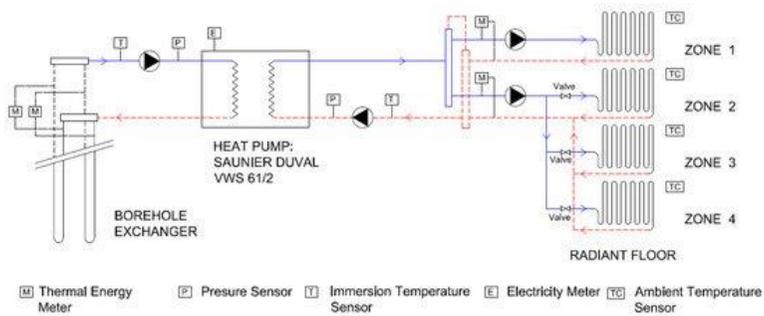


Figure 2: Ground-coupled heat pump (GCHP) system schematic.

The monthly energy demand for the building in heating and cooling modes is illustrated in Figure 3.

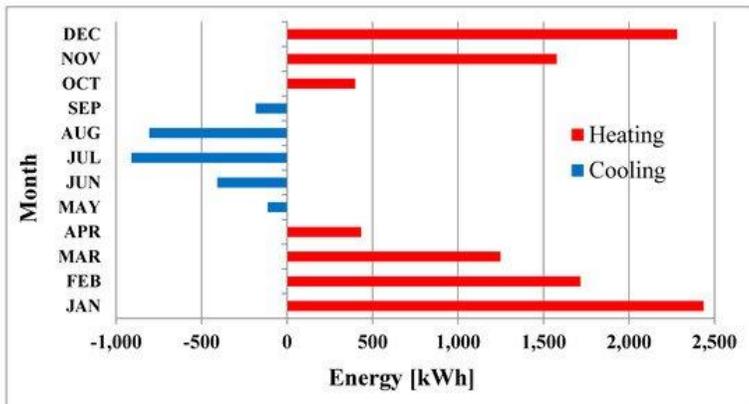


Figure 3: Monthly energy demand.

Borehole Heat Exchanger

The GHE is a vertical borehole with double U-shaped HDPE 100 pipes. The borehole was completely backfilled with bentonite, cement, and sand. The average thermal conductivity of the ground was 1.80 W/mK. The main thermal parameters are listed in Table 1.

Table 1: Soil features.

Soil Contents GRAVEL + CLAY + GRANITE
Thermal conductivity (W/mK) 1.8
Thermal diffusivity (m ² /d) 0.065
Average Thermal resistivity (K/(W/m) 0.059

Ground-Coupled Heat Pump Unit

Due to the importance of electricity consumption in the COP_{sys} of the installation, it is necessary to select the adequate heat pump capacity and ensure that it can supply the thermal loads of the building under analysis. The installed GCHP is an SD VWS 61/2 model manufactured by SAUNIER DUVAL. The VWS series uses a scroll compressor unit with R-407c as a refrigerant. The COP for heating is 4.2 and an EER of 3.92 for cooling. The VWS 61/2 model has a nominal heating capacity of 6 kW and a nominal cooling capacity of 4.9 kW (a rated power of 1.25 kW). An ethylene glycol aqueous solution is circulated between the heat pump and the BHE with a WILO TOP-S 30/4EM water circulating pump with a maximum flow capacity of 4 m³/h. The same model of water pump is used for the radiant floor system. The total annual energy consumption of the water recirculating pumps is 608 kWh, which represented the 25.65% of the total energy consumption of the GCHP system.

Figure 4 shows the power consumption and heating and cooling capacities, and they were used to model the heat pump in EnergyPlus. The nominal rates have been calculated using hot water at 35 °C and cold water at 7 °C for cooling capacity, and cold water at 5 °C and hot water at 45 °C for heating capacity.

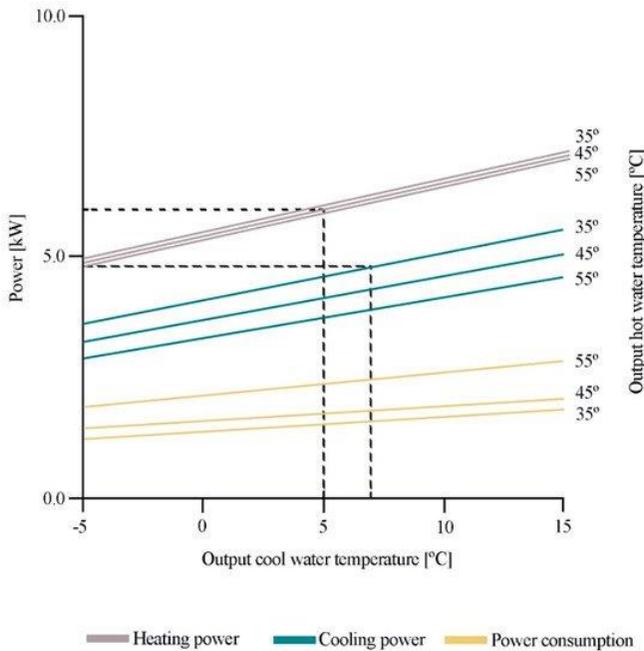


Figure 4: Heat pump performance data.

Mechanical Ventilation

The installed VMC is a SIBER DF MAX double flow equipment, with a maximum flow rate of 220 m³/h. The outside air is filtered and heated before being blown into the rooms in winter and cooled in summer with a 92% heat recovery.

Radiant Floor System

The RF system is distributed in four areas and with seven hydraulic circuits in order to ensure hydronic balance. The heat in each area is provided by one variable circulating pump and other variable circulation pumps for the other three zones. To achieve higher performances of the HVAC system, the thermostat controls the speed of the variable circulation pump depending on the heat carrier demand in the zone. Figure 5 shows the distribution of RF circuits over the ground floor. This system guarantees a homogeneous indoor temperature

distribution and increases the COP_{SYS} of the GCHP [19,20,21]. The main characteristics are listed in Table 2.

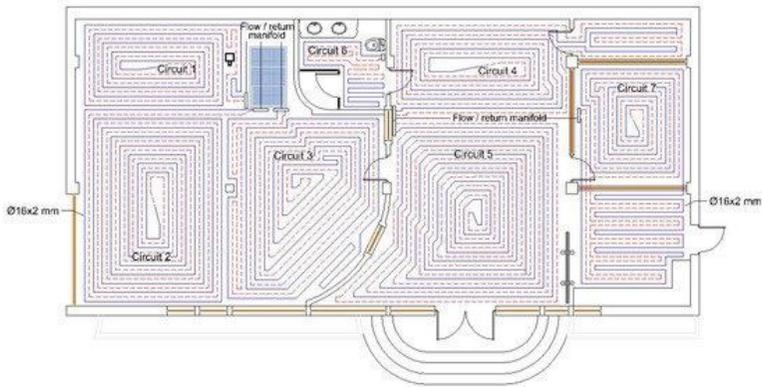


Figure 5: Radiant floor system.

Table 2: Characteristics of the radiant floor.

Active Radiant Floor Layer	
Pipe wall conductivity (W/mK)	0.37
Pipe wall thickness (m)	0.002
Pipe outside diameter (m)	0.016
Pipe spacing (m)	0.15
Specific heat coefficient fluid (kJ/kgK)	4.18

Data Acquisition and Experimental Procedure

A temperature and humidity sensors system were installed in the building, connected to a thermostat model EXACONTROL E7R C/SH, Saveris V2.0 converter, with a range of 0 to 80 °C and humidity range of 0% to 100%, with accuracy ± 0.15 °C and 3%, respectively. These temperature sensors are Pt100. A Siemens Coriolis mass flow meter, model SITRANS FC MASS 6000 with an accuracy $< 0.1\%$ was installed. The power meters are from Fluke, model 1736/EUS, with accuracy ± 1.4 of the nominal value and used to measure the power consumption of the HP, MV, internal/external pumps, PCs, and lighting. The temperature sensors for the RF system are Orkli V05, with a range of 0–100 °C with an accuracy of ± 0.3 °C.

Experiment Results

The study was carried out considering the two most unfavorable weeks in heating and cooling modes, obtained from the weather data file. The period corresponds to the months of January and July of 2011. The daily analysis period corresponds to the time band from 09:00 hs to 18:00 hs. Annual indoor and outdoor temperatures were measured for each zone and its evolution during the year is presented in Figure 6. It is observed that in heating and cooling modes, the average temperature for each zone reaches the defined setpoints of 22 °C in heating and 25 °C in cooling operation.

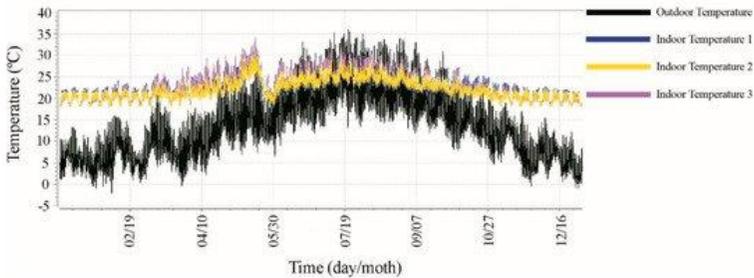


Figure 6: Recorded annual indoor and outdoor temperature.

Figure 7 and Figure 8 show the comparison between indoor/outdoor ambient air temperature and surface average floor temperature reached in heating and cooling modes. The results show that the GCHP system can maintain the indoor temperature during all the processes.

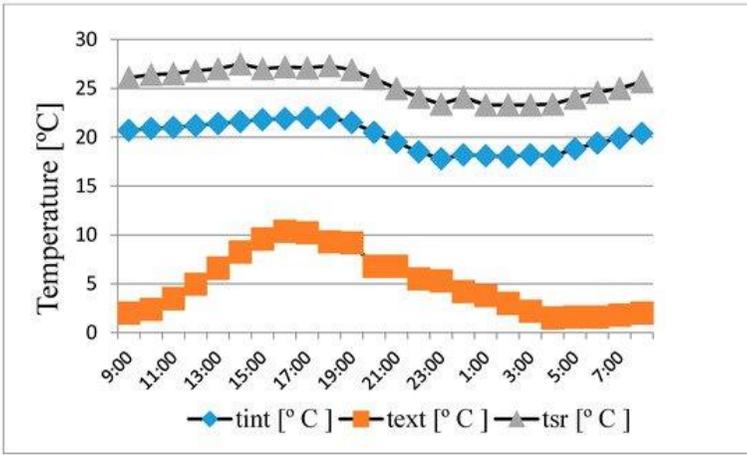


Figure 7: Indoor temperature in heating.

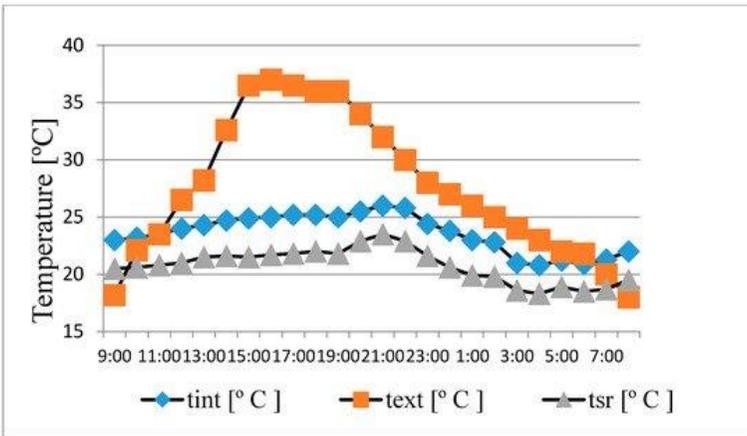


Figure 8: Indoor temperature in cooling.

Table 3 shows how the comfort temperature and RH parameters are reached in heating and cooling modes in a period of one year. In order to optimize the energy consumption, the DHW values are maintained at a low step. The COP_{sys} in cooling operation is higher than in heating. This is due to the MV in night periods exerting a cooling effect over the system, minimizing the thermal demand.

Table 3: Average values of experimental results. HVAC.

HVAC System	Text (°C)	Tint (°C)	HR (%)	Tsr (°C)	Thw (°C)	COP _{sys}	COP/ERR
Heating mode	6.69	21.7	36.8	26.89	30.24	3.86	4.85
Cooling mode	25.57	24.8	40.1	21.3	28.62	5.29	6.65

Figure 9 shows the non-existence of condensation.

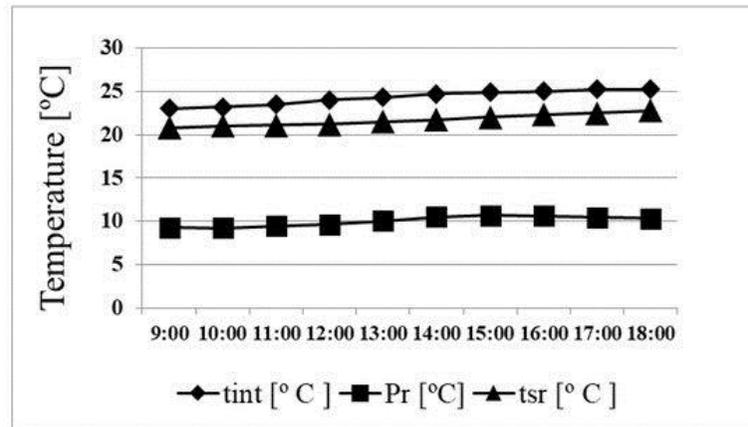


Figure 9: Radiant floor (RF) system temperatures.

Numerical Simulation

EnergyPlus was the software used for the design of the simulation model. Figure 10 shows the model of the building created in SketchUp, necessary to characterize the building.

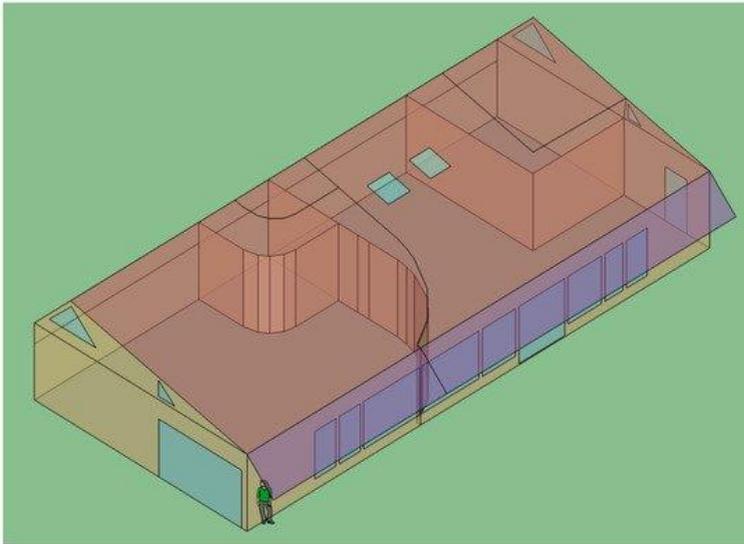


Figure 10: SketchUp plan.

Simulation Models in Energy Plus

EnergyPlus was the software selected to simulate the thermal operation of the office building. Using software specifications and building characteristics data, the calculation model is implemented. Three principal models are used in the simulation process. The heat exchanger was done using GLHEPRO program [22]. Based on physical parameters generated from the catalog data of the pump unit, a second model was used to describe the heat pump unit [23,24]. For modeling the transient heat conduction through the radiant floor system, the conduction transfer method (CTFs) was used.

Figure 11 shows the node scheme which represents the configuration of the system implemented in EnergyPlus to

integrate all the process and elements to model the simulation process.

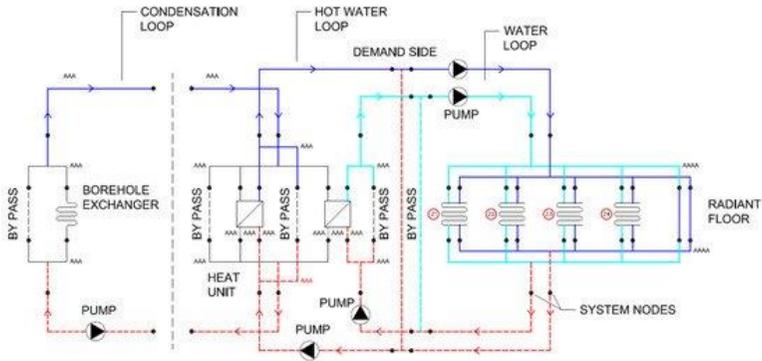


Figure 11: System nodes scheme.

Figure 12 and Figure 13 show the comparison between the simulated and experimental values using the statistical method coefficient of multiple determinations (R^2).

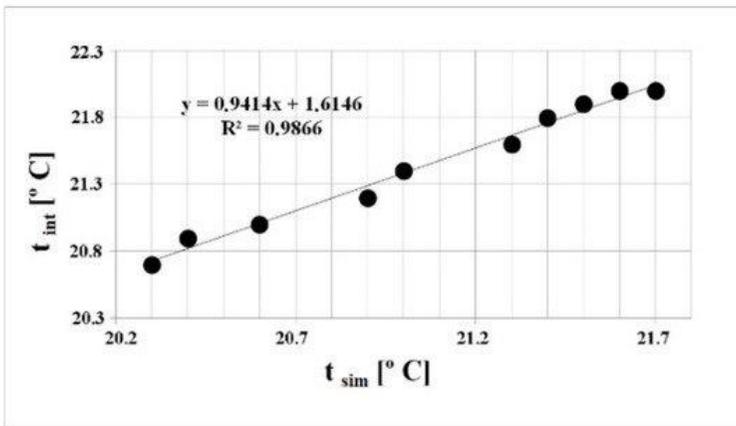


Figure 12: R^2 graph results in heating mode.

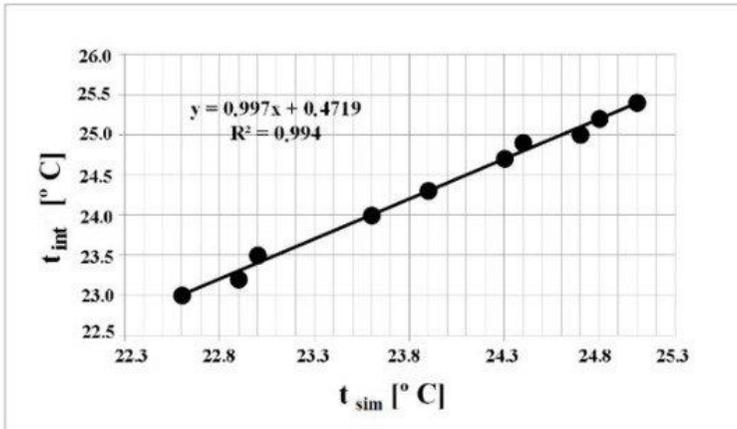


Figure 13: R² graph results in cooling mode.

Results and Discussion

In this section, the main results from real and simulated process are shown and analyzed. First, the most significant results of the GCHP systems are discussed, comparing the experimental and simulated results for the monthly average temperature of each of the four zones. The energy consumption of the system is illustrated comparing the experimental and simulated values obtained in the process. Finally, the heat flow transfer to the geothermal field and radiant floor system is analyzed and discussed.

In Figure 14, the comparison between the real and simulated average monthly zone temperature values are represented. It is observed that the HVAC system supplies the energy necessary to maintain the comfort temperature conditions of 22 °C in heating and 25 °C in cooling mode in all zones.

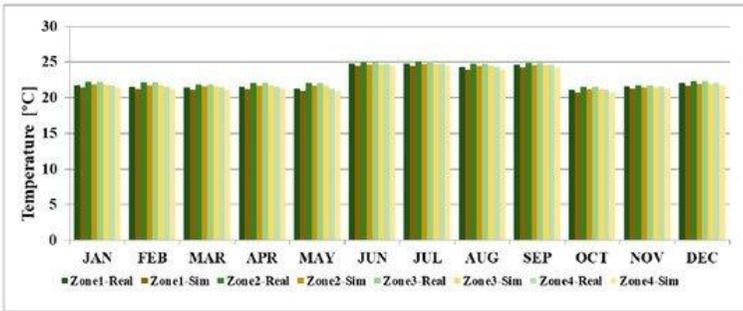


Figure 14: Experimental and simulated average monthly temperature for each zone.

The statistical method of coefficient of multiple determinations (R^2) was used to compare simulated and real values. The R^2 values for the different zones and the entire building are shown in Table 4. Zone 1 and 4 have lower R^2 values than other areas because these areas have peak demand to be areas that are not continually used (test room and bathroom), which translates into greater variability in the peak value of internal loads, therefore, the time response is longer such that the difference between the real and simulated values is higher. The other two zones have a higher linearity in their internal loads to be continuous use areas. As is shown in Figure 14, the zones 2 and 4 have R^2 values similar to the entire building.

Table 4: R^2 values for four zones and entire building.

T^a	R^2
T^a average, zone 1	0.9841
T^a average, zone 2	0.9913
T^a average, zone 3	0.9914
T^a average, zone 4	0.9871
T^a average, building	0.991

The consumption of the heat pump unit, circulating pumps, and auxiliary elements are considered to have the real value of the energy consumption of the building. In Figure 15, the experimental and simulated values of the total energy consumption are presented. The results are in accordance with monthly energy demand presented in Figure 3. The highest energy consumption corresponds to the heating period and is

higher than in cooling period. Using the same statistical method, an R^2 value of 0.9841 is obtained for the comparison of the energy consumption of the GCHP system in real and simulated conditions.

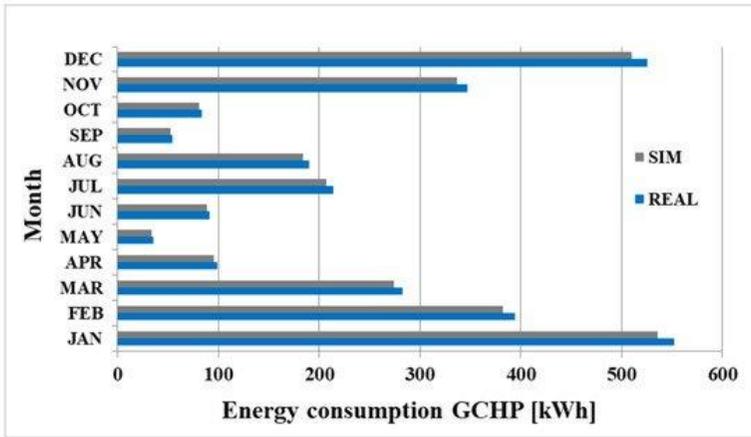


Figure 15: Energy consumption of the GCHP system.

Figure 16 shows the average monthly energy exchanged between the soil and the heat pump unit. It should be noted that this energy is more important in the period in which the temperature difference between the demand and the soil temperature is higher. These results guarantee a better performance of the heat pump and, therefore, a decrease of the energy consumption.

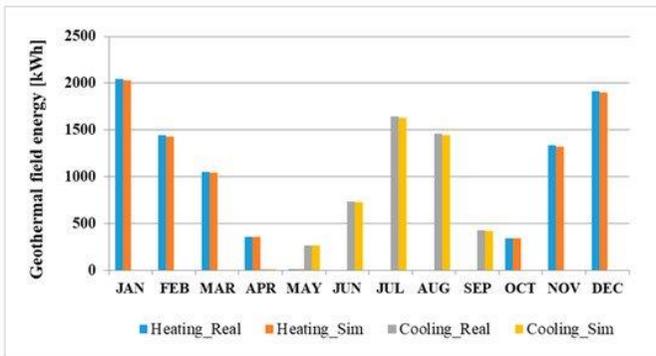


Figure 16: Monthly energy transferred to geothermal field.

In this case, the radiant floor can handle the sensible load of the different zones. In particular, we can see that in the heating period, the energy exchanged is higher than in cooling mode, due to the thermal demand being more important in this period. The values shown in Figure 17 are in accordance with the thermal demand.

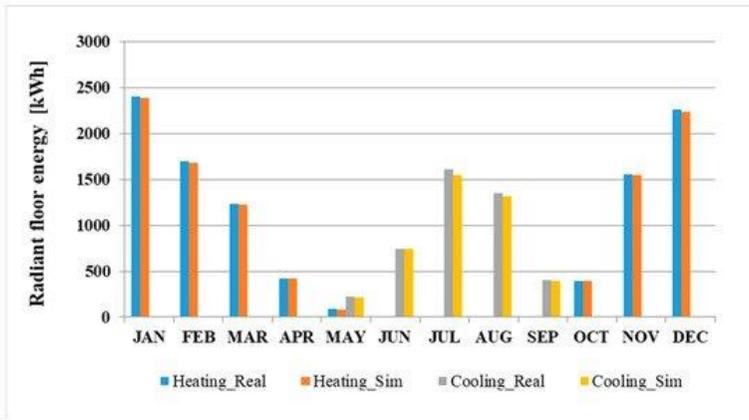


Figure 17: Monthly energy transferred to radiant floor system.

The simulations results are closer to the actual results in heating and cooling mode. The COP_{sys} in heating and cooling mode for experimental and simulated conditions is shown in Table 5.

Table 5: COP_{sys} real/simulated for heating and cooling operation.

	COP_{sys} Heating_ Real	COP_{sys} Heating_ Sim	COP_{sys} Cooling_ Real	COP_{sys} Cooling_ Sim
GCHP system	3.86	3.80	5.29	5.18

Comparison of COP and CO₂ Emissions with Other Proposed Technologies

In order to know the positioning of the GCHP system designed, a comparison with natural gas, diesel, and biomass was conducted. The three systems are composed of a boiler for each technology and radiant floor for distribution. An additional heat pump unit is installed in order to satisfy the cooling demand and maintain the

same conditions for the analysis. The model validated the calculated COP_{sys} and CO₂ emissions. In Table 6, the primary energy coefficients are listed in order to obtain the CO₂ emissions.

Table 6: Energy step factors.

Energy	To CO ₂ Emissions (kgCO ₂ /kWhEF)
Electricity	0.331
Diesel	0.311
Natural Gas	0.252
Biomass	0.018

In Table 7, annual COP_{sys} and CO₂ emissions values are shown.

Table 7: COP_{sys} and CO₂ emission results.

HVAC System	COP Heating Mode	COP Cooling Mode	CO ₂ Emissions (Kgs)
GCHP	3.86	5.29	985
Natural Gas	1.03	2.80	2741
Diesel	0.9	2.80	3715
Biomass	0.8	2.8	475

The results show that the GCHP system is the technology with the highest COP_{sys} in heating and cooling modes, and least CO₂ emissions, with the latter surpassed only by biomass.

Comparative Economic Analysis

For the comparative economic analysis of the four technologies proposed, the cost of the installations, maintenance, and energy source for each technology is presented. An economic lifetime of 15 years is considered. The cost data were obtained in 2015 through different companies specializing in these kinds of systems. Using the data of energy price forecast from the hydrocarbon geoportal of the Spanish industrial and energy ministry, and combined with the average annual fuel/electricity consumption obtained from real/simulated data, the annual fuel cost is calculated. A fixed percentage of the initial installation of

each system is considered for determining the annual maintenance cost. The results are shown in Table 8.

Table 8: Data used for economic analysis.

	Geothermal	Biomass	Natural gas	Diesel
Total Investment (€)	16,475	13,275	12,875	13,275
GCHP (€)	5800			
Boiler (€)		2800	2400	2800
Split (€)		4000	4000	4000
Borehole (€)	4325			
Radiant Floor (€)	6350	6475	6475	6475
Annual maintenance cost (%)	4.7%	4.0%	4.0%	4.0%
Cost effective (€/kWh)	0.1764	0.0507	0.0839	0.0905

As Figure 18 shows, GCHP systems are, for the first time, not the economically preferred investment as a HVAC system for buildings. Only after some years of functioning were the economic results for this technology better than all the other options. The accumulated savings (positive values) from GCHP versus natural gas, biomass, and diesel are shown in Figure 19.

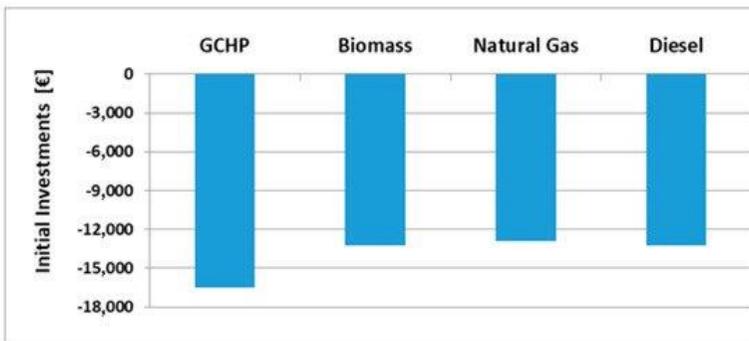


Figure 18: Initial investments for the technologies proposed.

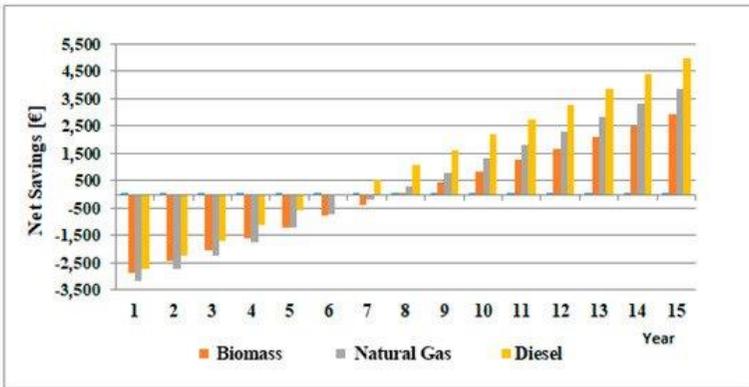


Figure 19: Annual net savings for a 15-year period.

After 6.5 years' functioning of the installation, the GCHP overcame the cost of diesel, and in the seventh year, natural gas. Only after 9 years did the proposed system reach costs equivalent to using biomass. In Figure 20, the accumulated cost for each technology during a 15-year period is illustrated.

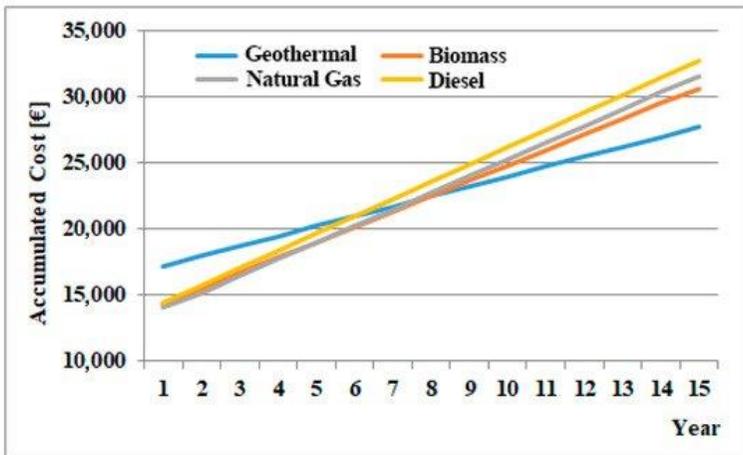


Figure 20: Accumulated cost for each technology.

Conclusions

The use of low enthalpy geothermal systems in buildings with low energy demand is a recommended option to satisfy HVAC and DHW necessities. This paper shows the position of the system proposed against other technologies—natural gas, diesel, and biomass—from energy, economic, and environmental aspects. The parameter selected to analyze the comfort of the installation is the mean indoor temperature in each area. EnergyPlus was the software used to create the simulation model, in order to describe the operation of the installation.

The results obtained show that the indoor temperature in each area is closed to the objective. The differences were only of 0.3/0.2 °C in heating and cooling modes respectively. The average temperature reached in cooling and heating modes was 24.8/21.7 °C, considered to be a good performance for the designed HVAC system.

A comparison between experimental and simulation results was done in order to validate the model designed. The R^2 results obtained represented a good approximation of the model to real operation. The performance of the system is represented by the COP_{sys} , which was 3.86/5.89 for heating and cooling modes, and represents a good performance.

Natural gas, diesel, and biomass were the technologies selected for the comparison done. The COP_{sys} obtained in heating modes was 1.03/0.9/0.8 respectively, which show the high performance of the GCHP compared with the rest, being higher between 3.7 and 4.8 times. In cooling operation, the COP_{sys} was two times higher because MV ensures the indoor air quality and exerts a cooling effect over the system, minimizing the thermal demand.

The economic analysis illustrates that the GCHP systems have a higher initial cost than the rest of the available technologies. After six and seven years, the GCHP maintains HVAC conditions at lower cost than diesel and natural gas, respectively, and reaches the same level as biomass in the ninth year.

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